

# SIR C.R.REDDY COLLEGE OF ENGINEERING

ELURU - 534 007



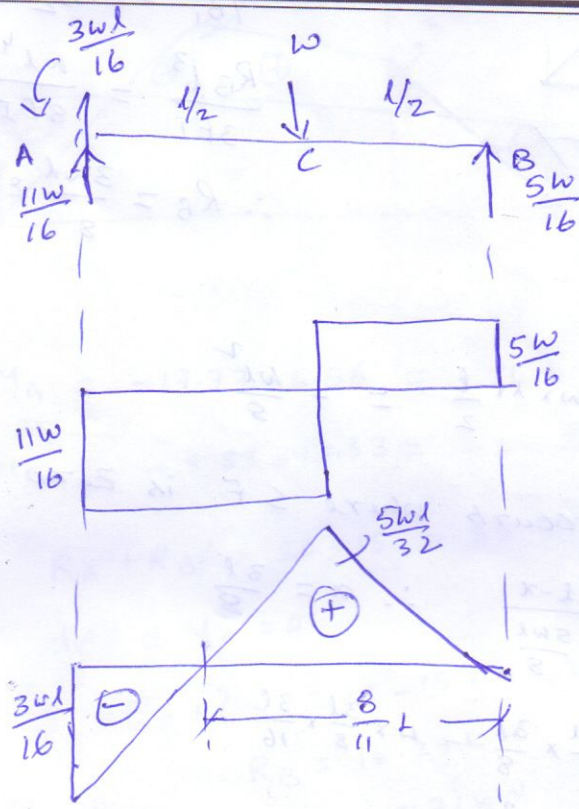
Regd. No.	A	N	S	N	E	R	S						
-----------	---	---	---	---	---	---	---	--	--	--	--	--	--

Class : II / IV B.Tech Branch : Civil Engg Date : 22 / 11 / 2021

Subject : Structural Analysis Signature of the Invigilator : .....

Marks Awarded :  Signature of the Subject Teacher : .....

1. A.



For no deflection at 'B'  
Deflection by reaction

$$y_{B1} = \frac{R_B l^3}{3EI}$$

Deflection by load 'w'

$$y_{B2} = \frac{W(\frac{l}{2})^3}{3EI} + \frac{W(\frac{l}{2})}{2EI} \times \frac{l}{2}$$

$$= \frac{Wl^3}{24EI} + \frac{Wl^3}{16EI}$$

$$y_{B2} = \frac{5Wl^3}{48EI}$$

$$y_{B1} = y_{B2}$$

$$\therefore \frac{R_B l^3}{3EI} = \frac{5Wl^3}{48EI} \quad \text{Then } R_B = \frac{5W}{16} \quad \& \quad R_A = \frac{11W}{16}$$

$$M_B = 0$$

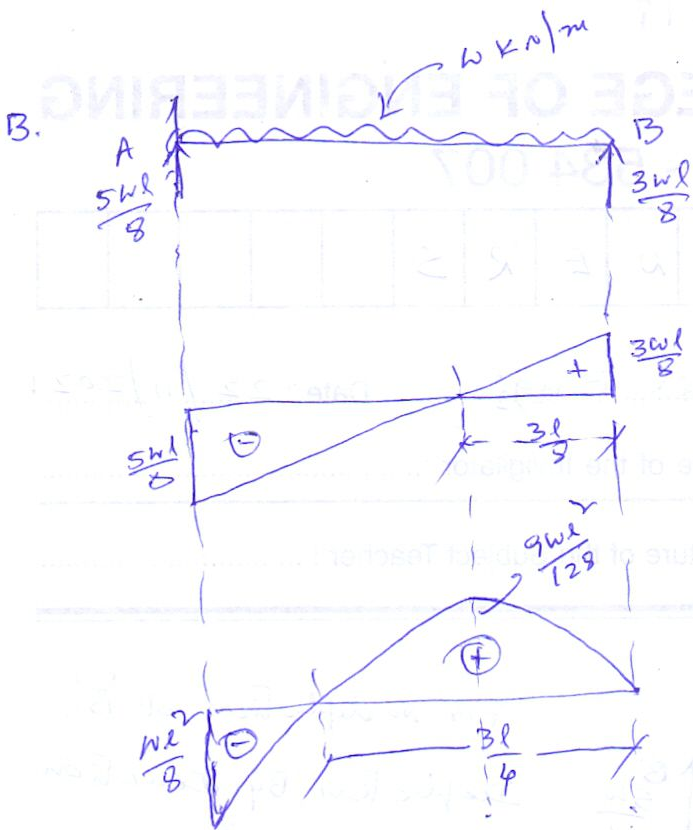
$$M_C = \frac{5W}{16} \times \frac{l}{2} = \frac{5Wl}{32}$$

$$M_A = \frac{5W}{16} \times l - W \times \frac{l}{2} = -\frac{3Wl}{16}$$

Point of contraflexure where BM changes sign or zero

$$\frac{5}{16} W \times x - W(x - \frac{l}{2}) = 0$$

$$\therefore x = \frac{8}{11} L \quad \text{from free end}$$



For no deflection at 'B'  
 Deflection By Reaction  $R_B$   

$$y_{B1} = \frac{R_B l^3}{3EI}$$

Deflection because of load  $w$

$$y_{B2} = \frac{wl^4}{8EI}$$

$$y_{B1} = y_{B2}$$

$$\frac{R_B l^3}{3EI} = \frac{wl^4}{8EI}$$

$$\therefore R_B = \frac{3wl}{8} \text{ \& } R_A = \frac{5wl}{8}$$

$$M_B = 0$$

$$M_A = \frac{3wl}{8} \times l - w \times l \times \frac{l}{2} = -\frac{wl^2}{8}$$

Maximum B.M. occurs where S.F. is zero

$$\frac{x}{\frac{3wl}{8}} = \frac{l-x}{\frac{5wl}{8}} \quad \therefore x = \frac{3l}{8}$$

$$M_{\text{max}} = \frac{3wl}{8} \times \frac{3l}{8} - w \times \frac{3l}{8} \times \frac{3l}{16}$$

$$= \frac{9wl^2}{128}$$

Point of contraflexure where B.M. is zero

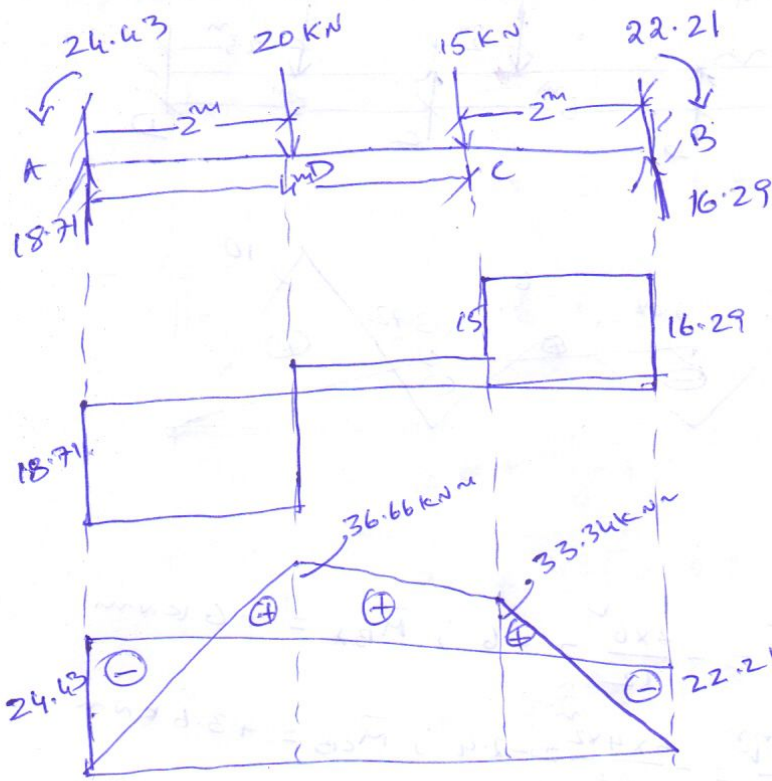
$$M_x = 0$$

$$\text{i.e. } \frac{3wl}{8} x - w \cdot x \cdot \frac{x}{2} = 0$$

$$\therefore x = \frac{3l}{4} \text{ from free end}$$



2.A.



Applying principle of superposition

$$M_A = M_{A1} + M_{A2}$$

$$M_B = M_{B1} + M_{B2}$$

$$M_{A1} = \frac{20 \times 2 \times 4^2}{6^2} = -17.77$$

$$M_{B1} = \frac{20 \times 2 \times 4^2}{6^2} = -8.88$$

$$M_{A2} = \frac{15 \times 4 \times 2^2}{6^2} = -6.66$$

$$M_{B2} = \frac{15 \times 4 \times 2^2}{6^2} = 13.33$$

$$\therefore M_A = -17.77 - 6.66 = -24.43 \text{ kNm}$$

$$M_B = -8.88 - 13.33 = -22.21 \text{ kNm}$$

$$R_A + R_B = 20 + 15 = 35 \text{ kN}$$

$$\sum M_A = 0$$

$$R_B \times 6 - 22.21 - 15 \times 4 - 20 \times 2 + 24.43 = 0$$

$$R_B = 16.29 \text{ kN}$$

$$R_A = 18.71 \text{ kN}$$

Positive Bending Moments

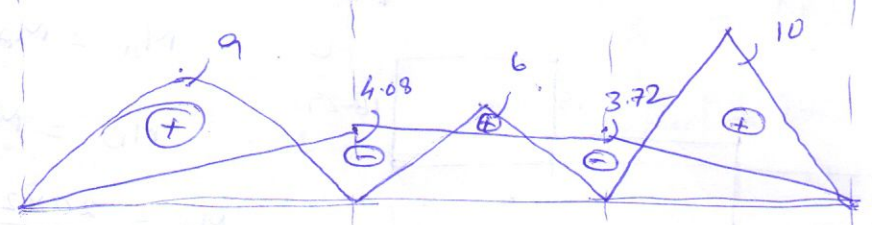
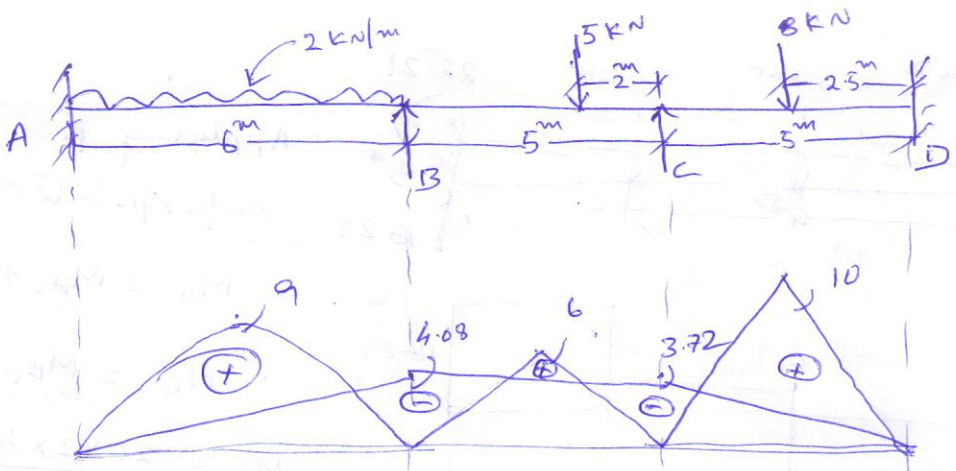
Assuming the beam as AB

$$R_A = 18.33 \text{ kN}; R_B = 16.67 \text{ kN}$$

$$M_C = + 16.67 \times 2 = 33.34 \text{ kNm}$$

$$M_D = + 18.33 \times 2 = 36.66 \text{ kNm}$$

3.



$$\bar{M}_{AB} = -\frac{wL^2}{12} = -\frac{2 \times 6^2}{12} = -6 ; \bar{M}_{BA} = +6 \text{ kNm}$$

$$\bar{M}_{BC} = -\frac{w_1 a b^2}{12} = -\frac{5 \times 4 \times 2^2}{12} = -2.4 ; \bar{M}_{CB} = +3.6 \text{ kNm}$$

$$\bar{M}_{CD} = -\frac{wl}{8} = -\frac{8 \times 5}{8} = -5 ; \bar{M}_{DC} = +5 \text{ kNm}$$

Slope Deflection Equations

$$M_{AB} = -6 + \frac{2EI}{6} (\theta_A + \theta_B) = -6 + \frac{EI}{3} \theta_B$$

$$M_{BA} = +6 + \frac{2EI}{6} (\theta_A + \theta_B) = +6 + \frac{2EI}{3} \theta_B$$

$$M_{BC} = -2.4 + \frac{2EI}{5} (\theta_B + \theta_C) = -2.4 + \frac{4EI\theta_B}{5} + \frac{2EI\theta_C}{5}$$

$$M_{CB} = +3.6 + \frac{2EI}{5} (\theta_B + \theta_C) = +3.6 + \frac{2EI\theta_B}{5} + \frac{4EI\theta_C}{5}$$

$$M_{CD} = -5 + \frac{2EI}{5} (\theta_C + \theta_D) = -5 + \frac{4EI\theta_C}{5}$$

$$M_{DC} = +5 + \frac{2EI}{5} (\theta_C + \theta_D) = +5 + \frac{2EI\theta_C}{5}$$

Equilibrium conditions

$$M_{BA} + M_{BC} = 0 \text{ and } M_{CB} + M_{CD} = 0$$

$$+6 + 0.66EI\theta_B - 2.4 + 0.8EI\theta_B + 0.4EI\theta_C = 0$$

$$1.16EI\theta_B + 0.4EI\theta_C = -3.6 \quad \text{--- (1)}$$

$$+3.6 + 0.4EI\theta_B + 0.8EI\theta_C - 5 + 0.8EI\theta_C = 0$$

$$0.4EI\theta_B + 1.6EI\theta_C = +1.4 \quad \text{--- (2)}$$

Solving Equations (1) & (2)

$$\text{We get } \theta_B = -\frac{2.89}{EI} ; \theta_C = \frac{1.59}{EI}$$

Substituting  $\theta_B$  &  $\theta_C$  values, in Slope Deflection Equations

$$M_{AB} = -6 + \frac{EI}{3} \times -2.89 = -6.96 \text{ KNm}$$

$$M_{BA} = +6 + \frac{2EI}{3} \times -2.89 = +4.08 \text{ KNm}$$

$$M_{BC} = -2.4 + \frac{4EI}{5} \times -2.89 + \frac{2EI}{5} \times 1.59 = -4.08 \text{ KNm}$$

$$M_{CB} = +3.6 + \frac{2EI}{5} \times -2.89 + \frac{4EI}{5} \times 1.59 = +3.72 \text{ KNm}$$

$$M_{CD} = -5 + \frac{4EI}{5} \times 1.59 = -3.72 \text{ KNm}$$

$$M_{DC} = +5 + \frac{2EI}{5} \times 1.59 = +5.636 \text{ KNm}$$

Positive Bending Moments

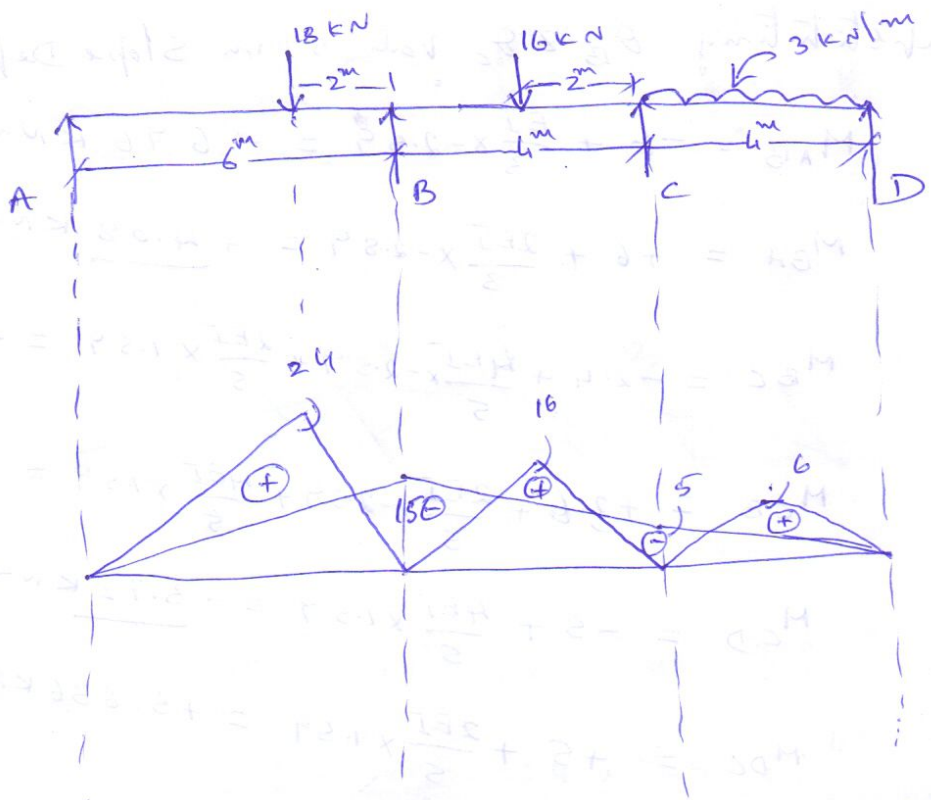
$$M_{AB} = + \frac{wl^2}{8} = \frac{2 \times 6^2}{8} = +9 \text{ KNm}$$

$$M_{BC} = + \frac{w_1 l_1}{1} = \frac{5 \times 2 \times 2}{5} = +6 \text{ KNm}$$

$$M_{CD} = + \frac{wl}{4} = \frac{8 \times 5}{4} = +10 \text{ KNm}$$



4.



$$DF_{BA} = \frac{3EI/6}{3EI/6 + 4EI/4} = 0.33; \quad DF_{BC} = 0.67$$

$$DF_{CB} = \frac{4EI/4}{4EI/4 + 3EI/4} = 0.57; \quad DF_{CD} = 0.43$$

FEMs

$$\bar{M}_{AB} = - \frac{18 \times 4 \times 2}{6^2} = -8 \text{ KNm}$$

$$\bar{M}_{BA} = + \frac{18 \times 4 \times 2}{6^2} = 16 \text{ KNm}$$

$$\bar{M}_{BC} = - \frac{wl}{8} = \frac{16 \times 4}{8} = -8 \text{ KNm}; \quad \bar{M}_{CB} = +8 \text{ KNm}$$

$$\bar{M}_{CD} = - \frac{wl^2}{12} = - \frac{3 \times 4^2}{12} = -4 \text{ KNm}; \quad \bar{M}_{DC} = +4 \text{ KNm}$$

Joint	A	B		C		D
Member	AB	BA	BC	CB	CD	DC
DF	-	0.33	0.67	0.57	0.43	-
FEM <sub>0</sub>	-8	+16	-8	+8	-4	+4
ROM	+8	-	-	-	-	-4
COM	-	+4	-	-	-2	-
IM	0	+20	-8	+8	-6	0
DM	-	-3.96	-8.04	-1.14	-0.86	-
COM	-	-	-0.57	-4.02	-	-
DM	-	+0.1881	+0.3819	+2.2914	+1.7286	-
COM	-	-	+1.1457	+0.1909	-	-
DM	-	-0.3780	-0.7676	-0.1088	-0.082	-
FM	0	+15.853	-15.853	+5.2135	-5.2135	-

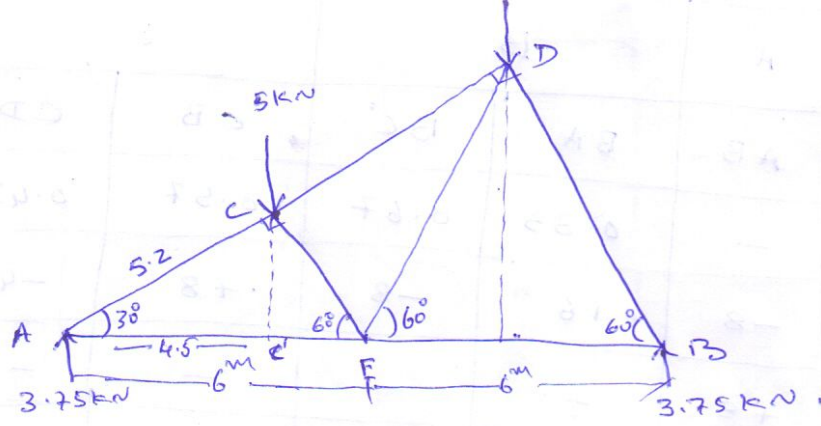
+ve B.M.

$$\text{on Span AB} = \frac{w a b^2}{l} = \frac{18 \times 2 \times 4}{6} = 24 \text{ kNm}$$

$$\text{Span BC} = \frac{w l}{4} = \frac{16 \times 4}{4} = 16 \text{ kNm}$$

$$\text{Span CD} = \frac{w l^2}{8} = \frac{3 \times 4^2}{8} = 6 \text{ kNm}$$

5.



To find out the reactions

$$AC = AE \cos 30^\circ = 6 \cos 30^\circ = 5.2 \text{ mtr}$$

$$\therefore AC' = AC \cos 30^\circ = 5.2 \cos 30^\circ = 4.5 \text{ mtr}$$

$$\sum M_A = 0$$

$$R_B \times 12 - 2.5 \times 9 - 5 \times 4.5 = 0$$

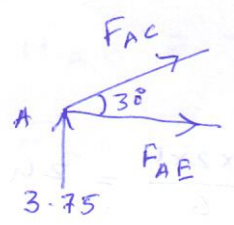
$$\therefore R_B = 3.75 \text{ kN}$$

$$R_A + R_B = 5 + 2.5 = 7.5 \text{ kN}$$

$$\therefore R_A = 3.75 \text{ kN}$$

By Method of Joints

Joint 'A'



$$\sum V = 0$$

$$3.75 + F_{AC} \sin 30^\circ = 0$$

$$\therefore F_{AC} = \frac{-3.75}{\sin 30^\circ} = -7.5 \text{ (C)}$$

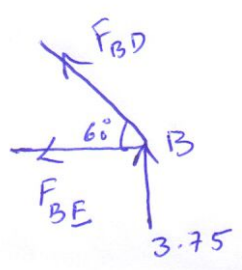
$$\sum H = 0$$

$$F_{AE} + F_{AC} \cos 30^\circ = 0$$

$$\therefore F_{AE} = -F_{AC} \cos 30^\circ = -(-7.5 \cos 30^\circ)$$

$$F_{AE} = +6.495 \text{ (T)}$$

Joint 'B'



$$\sum V = 0$$

$$3.75 + \sin 60^\circ F_{BD} = 0$$

$$\therefore F_{BD} = \frac{-3.75}{\sin 60^\circ} = -4.33 \text{ (C)}$$

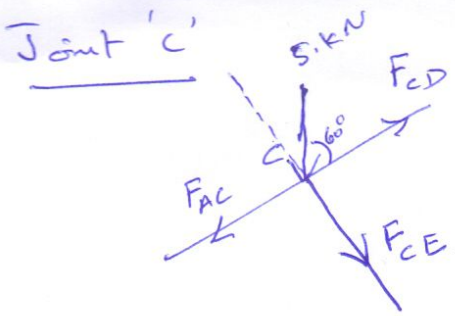
$$\sum H = 0$$

$$F_{BE} + F_{BD} \cos 60^\circ = 0$$

$$F_{BE} = -F_{BD} \cos 60^\circ$$

$$\therefore F_{BE} = -(-4.33 \cos 60^\circ) = +2.165 \text{ (T)}$$





$$\sum V = 0$$

$$F_{CE} + 5 \sin 60^\circ = 0$$

$$\therefore F_{CE} = -5 \sin 60^\circ = -4.33 (C)$$

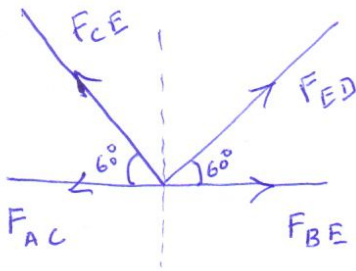
$$\sum H = 0$$

$$F_{CD} - F_{AC} - 5 \cos 60^\circ = 0$$

$$F_{CD} = F_{AC} + 5 \cos 60^\circ = -7.5 + 2.5$$

$$\therefore F_{CD} = -5 (C)$$

Joint 'E'



$$\sum V = 0$$

$$F_{CE} \sin 60^\circ + F_{ED} \sin 60^\circ = 0$$

$$\therefore F_{ED} = -F_{CE}$$

$$= -(-4.33) =$$

$$= +4.33 (T)$$

SN	Member	Magnitude of FORCE	Nature of FORCE
1	AE	6.495 N	T
2	AC	7.5 N	C
3	BE	2.165 N	T
4	BD	4.33 N	C
5	CE	4.33 N	C
6	CD	5.0 N	C
7	ED	4.33 N	T